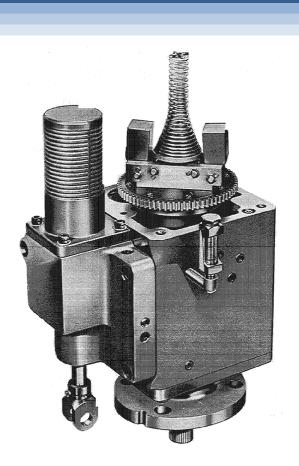


Product Manual 36600 (Revision H, 5/2015) Original Instructions



PG Governor Basic Elements

Installation and Operation Manual



Read this entire manual and all other publications pertaining to the work to be performed before installing, operating, or servicing this equipment.

Practice all plant and safety instructions and precautions.

Failure to follow instructions can cause personal injury and/or property damage.



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Any unauthorized modifications to or use of this equipment outside its specified mechanical, electrical, or other operating limits may cause personal injury and/or property damage, including damage to the equipment. Any such unauthorized modifications: (i) constitute "misuse" and/or "negligence" within the meaning of the product warranty thereby excluding warranty coverage for any resulting damage, and (ii) invalidate product certifications or listings.



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Warnings and Notices

Important Definitions



This is the safety alert symbol. It is used to alert you to potential personal injury hazards. Obey all safety messages that follow this symbol to avoid possible injury or death.

- **DANGER**—Indicates a hazardous situation which, if not avoided, will result in death or serious injury.
- **WARNING**—Indicates a hazardous situation which, if not avoided, could result in death or serious injury.
- **CAUTION**—Indicates a hazardous situation which, if not avoided, could result in minor or moderate injury.
- **NOTICE**—Indicates a hazard that could result in property damage only (including damage to the control).
- **IMPORTANT**—Designates an operating tip or maintenance suggestion.

WARNING Overspeed / Overtemperature / Overpressure	The engine, turbine, or other type of prime mover should be equipped with an overspeed shutdown device to protect against runaway or damage to the prime mover with possible personal injury, loss of life, or property damage. The overspeed shutdown device must be totally independent of the prime mover control system. An overtemperature or overpressure shutdown device may also be needed for safety, as appropriate.
WARNING Personal Protective Equipment	The products described in this publication may present risks that could lead to personal injury, loss of life, or property damage. Always wear the appropriate personal protective equipment (PPE) for the job at hand. Equipment that should be considered includes but is not limited to:

- Eye Protection
- Hearing Protection
- Hard Hat
- Gloves
- Safety Boots
- Respirator

Always read the proper Material Safety Data Sheet (MSDS) for any working fluid(s) and comply with recommended safety equipment.

WARNING Start-up

Be prepared to make an emergency shutdown when starting the engine, turbine, or other type of prime mover, to protect against runaway or overspeed with possible personal injury, loss of life, or property damage.



On- and off-highway Mobile Applications: Unless Woodward's control functions as the supervisory control, customer should install a system totally independent of the prime mover control system that monitors for supervisory control of engine (and takes appropriate action if supervisory control is lost) to protect against loss of engine control with possible personal injury, loss of life, or property damage.

NOTICE

To prevent damage to a control system that uses an alternator or battery-charging device, make sure the charging device is turned off before disconnecting the battery from the system.

Battery Charging Device

Electrostatic Discharge Awareness

NOTICE	Electronic controls contain static-sensitive parts. Observe the following precautions to prevent damage to these parts:
Electrostatic Precautions	 Discharge body static before handling the control (with power to the control turned off, contact a grounded surface and maintain contact while handling the control). Avoid all plastic, vinyl, and Styrofoam (except antistatic versions) around printed circuit boards. Do not touch the components or conductors on a printed circuit board with your hands or with conductive devices. To prevent damage to electronic components caused by improper handling, read and observe the precautions in Woodward manual 82715, Guide for Handling and Protection of Electronic Controls, Printed Circuit Boards, and Modules.

Follow these precautions when working with or near the control.

- 1. Avoid the build-up of static electricity on your body by not wearing clothing made of synthetic materials. Wear cotton or cotton-blend materials as much as possible because these do not store static electric charges as much as synthetics.
- 2. Do not remove the printed circuit board (PCB) from the control cabinet unless absolutely necessary. If you must remove the PCB from the control cabinet, follow these precautions:
 - Do not touch any part of the PCB except the edges.
 - Do not touch the electrical conductors, the connectors, or the components with conductive devices or with your hands.
 - When replacing a PCB, keep the new PCB in the plastic antistatic protective bag it comes in until you are ready to install it. Immediately after removing the old PCB from the control cabinet, place it in the antistatic protective bag.

Chapter 1. PG Governor Basic Elements

Introduction

This manual is one of several covering various components of the type PG governor, and can be combined with other "sectional" manuals to describe fully a particular PG governor model.

Woodward PG governors control the speed of diesel, gas, and dual fuel engines and steam turbines used in a variety of fields. They find service on engines and turbines driving pumps, compressors, alternators, variable speed dc generators, marine propulsion units, and papermaking machines.

In addition to the primary function of controlling engine or turbine speed, the governor is often required to perform such auxiliary functions as limiting engine load, varying engine load to maintain a constant engine power output for a given speed setting, shutting the engine down in the event of lubricating oil pressure failure, etc. Each governor is designed to meet the needs of the engine and the operating requirements of the installation in which it is used.

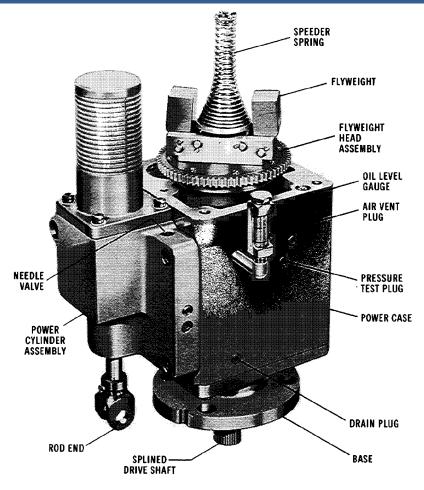
All PG governors have similar basic elements regardless of how simple or complex the complete control may be. The following elements, found in each PG governor, are sufficient to enable the governor to maintain a constant engine speed as long as the load does nor exceed engine capacity:

- an oil pump, storage area for oil under pressure, and a relief valve by which maximum oil pressure may be limited;
- a centrifugal flyweight head-pilot valve assembly which controls flow of oil to and from the governor power cylinder assembly;
- a power cylinder assembly—sometimes referred to as a servomotor—which repositions the fuel racks, fuel valve, or steam valve of the engine or turbine;
- a compensating system which gives stability to the governed system;
- a means of adjusting the governor (and thus, engine) speed setting.

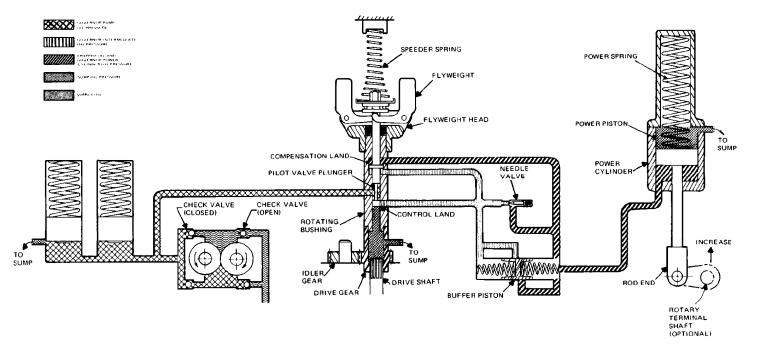
Either of two kinds of power cylinder assemblies can be used: a spring-loaded assembly or a differential assembly.

This manual describes the operation of the basic elements of a PG governor having a spring-loaded power cylinder assembly. Manual 36602 describes the arrangement having a differential power cylinder assembly.

Figure 1-2 shows the schematic arrangement of these elements in the relative positions they assume when the engine is operating on-speed under steady-state conditions. While differences exist in the actual design details of these elements from one governor to another, the scheme of operation is the same in each.









Governor Oil Pump

Each PG governor contains its own oil sump and oil pump. The governor drive shaft, driven at a speed proportional to engine speed by a mechanical connection to the engine, rotates the pump drive gear and the governor pilot valve bushing. As the rotating drive gear turns the idler gear, oil is drawn from the oil sump and is carried in the space between the gear teeth and the walls of the gear pocket to the discharge side of the pump. The oil is forced from the space between the gear teeth as the drive and idler gears mesh.

Assume that all control valves of the governor are closed. Oil deposited on the discharge side of the pump first fills the various oil passages, and then forces the accumulator pistons up against the downward force of the accumulator springs. When the piston uncovers the bypass hole, the excess oil from the governor pump returns to sump. The accumulators thus not only provide a reservoir for pressure oil but also act as a relief valve to limit maximum pressure in the hydraulic circuit.

The arrangement of the four check valves on the suction and discharge sides of the oil pump permits the governor drive shaft to be rotated in either direction without any changes being made in or to the governor. The direction of pump rotation does not affect the oil pressure system or governor operation. Were the pump gears rotated in the directions opposite those shown in Figure 1-2, the open check valves would be closed and the closed check valves opened.

(Some governor models are built without check valves. In these units, two plugs replace the two closed check valves so that the governor must be rotated in one direction only.)

Flyweight Head–Pilot Valve Assembly

The pilot valve plunger moves up and down in the rotating pilot valve bushing to control the flow of oil to or from the power cylinder assembly. When the pilot valve plunger is centered (the control land of the plunger exactly covers the control port of the bushing), no oil flows to or from the power cylinder assembly.

The greater of two forces moves the pilot valve plunger up or down. The centrifugal force developed by the rotating flyweights is translated into an upward force which tends to lift the plunger. The centrifugal force is opposed by the downward force of the speeder spring. When the opposing forces are equal, the pilot valve plunger is stationary.

With the pilot valve plunger centered and the engine running on-speed, a change in either of the two forces will move the plunger from its centered position. The plunger will be lowered (1) if the governor speed setting is unchanged but an additional load slows the engine and governor (thereby decreasing the centrifugal force developed by rotating flyweights) or (2) if the engine speed is unchanged but the speeder spring force is increased to raise the governor speed setting. Similarly, the pilot valve plunger will be raised (1) if the governor speed setting is unchanged but load is removed from the engine causing an increase in engine and governor speed (and hence, an increase in the centrifugal force developed by the rotating flyweights), or (2) if the engine speed is unchanged but the speeder spring force is reduced to lower the governor speed setting.

The thrust bearing atop the ballarm toes permits the pilot valve bushing to rotate while the pilot valve plunger does nor rotate. In this way, static friction between the bushing and plunger is minimized.

There are several styles of flyweight head assemblies available. The exact model used in any one governor depends on the application.

A "solid" head assembly is used in governors on prime movers which afford a smooth drive to the governor.

"Spring driven" and "spring driven, oil damped" head assemblies are used to filter torsional vibrations which may be imparted to the governor by the drive from the engine. (These torsional vibrations may originate from a source other than the drive itself but reach the governor through the drive connection.) Unless minimized or eliminated, the flyweight head will sense these torsional vibrations as speed changes and continually adjust the fuel valve or racks in an attempt to maintain a constant speed.

Spring-Loaded Power Cylinder Assembly

The governor pilot valve plunger controls the movement of the power piston in the power cylinder assembly. The power piston, acting through the connecting linkage, controls the engine or turbine fuel.

The power spring continually urges the power piston down in the "decrease fuel" direction. However, the power piston will not move down unless the pilot valve plunger is raised above its centered position. Only when the pilot valve plunger is above center can the oil trapped in the circuit between the plunger and power piston escape to sump. If the pilot valve plunger is lowered, pressure oil from the governor pump will be directed to the power piston and will push the piston up— against the force of the power spring—in the direction to increase fuel.

Note that the power piston will move only when the pilot valve plunger is uncentered to permit the oil flow required. With the plunger centered, the power piston is, in effect, hydraulically locked.

The output of the power cylinder assembly is normally a push-pull motion. Appropriate linkage within the power cylinder assembly can change the output to rotary motion. The power piston movement is the same in either case.

It should also be recognized that positioning the power cylinder assembly at an angle other than that shown in Figure 1-1 would have no effect on the assembly operation.

Compensating System

Stability of the governed system is achieved by the use of a temporary negative feedback signal which biases the speed signal to the pilot valve plunger. This temporary feedback signal is in the form of a pressure differential applied across the compensating land of the pilot valve plunger. The pressure differential is derived from the "buffer compensating system", and is dissipated as engine speed returns to the normal or set speed.

PG Governor Basic Elements

The buffer piston, buffer springs, and needle valve in the hydraulic circuit between the control land of the pilot valve plunger and the power piston comprise the "buffer compensating system" of the governor. Lowering the pilot valve plunger permits a flow of pressure oil into the buffer cylinder and power cylinder to move the power piston up to increase fuel. Raising the pilot valve plunger permits oil to flow from the buffer cylinder and power cylinder to the governor sump, and the power piston spring moves the power piston down to decrease fuel.

This flow of oil in the buffer system—in either direction—carries the buffer piston in the direction of flow, compressing one of the buffer springs and releasing the other. The buffer piston movement, increasing the loading on one buffer spring while decreasing the loading on the other, creates a slight difference in the oil pressures on the two sides of the buffer piston. The higher oil pressure is on the side of the piston opposite the spring being compressed. The difference in oil pressure is proportional to the buffer piston displacement, and is dependent on the rate of the buffer springs selected for the particular installation.

The oil pressure on one side of the buffer piston is transmitted to the lower side of the compensation land on the pilot valve plunger, the pressure on the other side of the piston is transmitted to the upper side of the compensation land. The difference in oil pressures produces a net force (often termed a "compensating force")—upward or downward, as the case may be—which assists the flyweights or speeder spring in re-centering the pilot valve plunger whenever a fuel correction is made.

Operation of the buffer system can be seen by following the sequence of operations when the engine speed slows down because of the addition of a relatively small load.

The decrease in centrifugal force developed by the rotating flyweights permits the speeder spring to push the flyweights in, lowering the pilot valve plunger and opening the control port.

As the buffer piston moves in the direction of the oil flow—from pilot valve to power cylinder—the right-hand buffer spring is compressed and the left-hand spring is relieved. The oil displaced by the buffer piston as is moves to the right forces the power piston up, thereby increasing fuel to the engine, and the engine begins to accelerate. The buffer piston moves to the right, moving the power piston up, until the upward force created by the pressure differential across the buffer piston and compensating land is sufficient, when added to the centrifugal force from the rotating flyweights, to re-center the pilot valve plunger. As soon as the pilot valve plunger is re-centered, the power piston movement stops. When the governor is properly adjusted, this new piston position corresponds to the fuel increase needed to operate the engine at a set speed with the new load even though the engine has nor yet returned to the see speed. As the engine continues to accelerate to the set speed, the centrifugal force developed by the rotating flyweights increases. To offset this increase in centrifugal force, it is necessary to reduce the net upward force resulting from the pressure differential across the compensation land as the centrifugal force increases. (Otherwise, the pilot valve plunger would be lifted above center and the power piston would move down to decrease fuel.) This is accomplished by equalizing the pressures on both sides of the compensation land as a rate proportional to the rate at which the engine speed returns to normal. As the pressure differential is decreased, the net compensating force is decreased. If the decrease in compensating force can be made as the same rate as the centrifugal force is increased, the pilot valve plunger will remain centered and the power piston will remain stationary. As the pressure differential is being dissipated, the buffer spring compressed returns the buffer piston to its normal, central position. When the pressure differential has been reduced to zero, the compensating force will be reduced to zero. At this time the engine should be back to its set speed, developing the centrifugal force necessary from the flyweights to keep the pilot valve plunger centered.

The rate at which the pressure differential is reduced to zero depends on the needle valve setting. The needle valve setting allows each governor to be "matched" to the engine on which it is used. The final setting of the needle valve cannot be made until the governor is installed on its engine.

When a relatively small load is removed from the engine while it is running onspeed under steady-state conditions, the sequence of events is similar but in the opposite directions.

The engine speed increases. The rotating flyweights develop additional centrifugal force and lift the pilot valve plunger. The oil between the buffer piston and the pilot valve plunger is connected to sump. The power spring pushes the power piston down, forcing the buffer piston towards the pilot valve plunger. A pressure differential again exists between the oil lines to the upper and lower sides of the compensating land of the pilot valve plunger.

The greater pressure is now on the upper side of the compensating land, tending to push the pilot valve plunger back down to its centered position.

When the compensating force becomes sufficient to offset the increase in centrifugal force, the pilot valve plunger is returned to its centered position even though the engine speed is still greater than the set speed. As before, the compensating force is again dissipated at a rate proportional to the rate at which the engine returns to normal speed so that the pilot valve plunger remains centered.

The sequence of movements within the governor occur almost simultaneously rather than in the step-by-step manner described.

Bypass passages are provided in the buffer cylinder to facilitate large corrective movements of the power piston. A large increase or decrease in speed setting, or a large increase or decrease in load on the engine, requires a correspondingly large movement of the power piston to make the necessary fuel correction. At such times, the buffer piston moves far enough to uncover the bypass port (to pressure or drain, depending on the correction being made).

The difference in oil pressures on the opposite sides of the buffer piston and the compensation land cannot exceed the difference which exists when the buffer piston uncovers the bypass port. With the bypass port uncovered, oil flows directly to or from the lower cylinder without further increasing the pressure differential existing on the compensation land.

The bypass ports permit the power piston to move quickly in response to large changes in load or speed setting. Since the pressure differential across the compensating land of the pilot valve plunger does nor continue to increase, the speed more than returns to normal (or to the new setting). This overshoot in speed is not large, and is tolerated in order to obtain the desired fast response. As soon as the speed changes beyond the desired point, normal governor action reverses the power piston movement, and stable operation is quickly established.

Surrounding the power piston and its piston rod are grooves connected to the intermediate oil pressure between the pilot valve and the buffer piston. These seal grooves—not shown in the schematic—have nothing to do with operation of the governor, but are used to ensure that any leakage of pressure oil from the power cylinder to the sump comes from a part of the hydraulic circuit where it will do no harm.

"Remote" power cylinder assemblies (assemblies which are located away from the governor power case) have three tubes connecting the power cylinder to the power case. These tubes carry (1) oil to and from the buffer, (2) drain oil from the power cylinder, and (3) oil to the power cylinder seal grooves. To eliminate the possibility of trapping air in the oil, the cylinder must be mounted below the governor oil level, and the connecting tubes must slope upward to the governor.

Speed Setting Adjustment

A number of speed setting arrangements are available for the PG governor. The exact method used depends upon the operating scheme of the installation in which it is used.

It is enough to observe at this point that no matter how simple or complex the means employed may be, the ultimate objective is to increase or decrease the compression of the speeder spring. An increase in the speeder spring compression requires that the engine run faster so that the rotating flyweights will develop the additional centrifugal force needed to re-center the pilot valve plunger. Conversely, a decrease in spring compression requires a slower engine speed.

The various speed setting arrangements are described in separate manuals.

The recommended speed range for the PG governor is 250 rpm to 1000 rpm with a maximum speed range of 200 rpm to 1600 rpm.

Chapter 2. Installation

General

At all times, use care in handling the governor; be particularly careful to avoid striking the drive shaft. Do not drop or rest the governor on its drive shaft. Such treatment could damage the governor drive components.

A gasket should be used between the mounting pad and the governor base when the governor is installed on the engine or turbine. The governor should be mounted squarely and the drive connection properly aligned. If the governor is equipped with a serrated or splined drive shaft, it should slip freely into the internal serrations or splines of the drive. If a keyed type governor drive shaft is used, the gear must slip on the shaft freely and should be checked to ensure that it meshes properly with its mating gear. The gears should run freely without binding or excessive backlash. Irregularities caused by uneven gear teeth, shaft runout, etc., will result in erratic governing.

Linkage Adjustment

The linkage from the governor to the fuel or steam control must be properly aligned. Any friction or lost motion must be eliminated. Adjustments of fuel linkage must provide for control of fuel from "OFF" to "FULL FUEL" within the limits of the stroke of the governor output connection. For specific information on fuel linkage installation, refer to the engine instruction manual.

When the governor has been properly mounted and the linkage connections completed, make the other connections (pneumatic, electrical, etc.) to the governor.

Oil Specifications

Information on oils for use in hydraulic governors is available in manual 25071. The recommended continuous operating temperature is 140 to 200 °F (60 to 93 °C). The ambient temperature limits are -20 to +210 °F (-29 to +99 °C). Contact Woodward if the temperature is beyond these limits. Hydraulic fluid pour point must be below the lowest expected starting temperature. Use SAE 20 or 30 oil for ordinary temperature conditions. If governor operating temperatures are extremely hot, use SAE 40 to 50; if extremely cold, use SAE 10. In moat cases the same oil that is used in the engine or turbine may be used in the governor.

Adjustments

Normally, the only requirements for putting a new or overhauled governor into service are filling the governor with oil and adjusting the compensation needle valve to obtain maximum stability. All other operating adjustments are made during factory testing according to engine manufacturer's specifications and should not require further adjustment. Do not attempt internal adjustment of the governor unless thoroughly familiar with the proper procedures.

Compensation Needle Valve Adjustment

The compensation needle valve is an adjustable part of the compensation system. Its setting, which directly affects governor stability, depends on the individual characteristics of the prime mover:

- 1. With the prime mover operating at IDLE, open the compensation needle valve several turns to cause the engine to hunt. In some cases, opening of the needle valve alone may not cause the engine to hunt, but manually disturbing the governor speed setting will. Allow several minutes of hunting to remove trapped air In the hydraulic circuits.
- Close the compensation needle valve gradually until hunting is just eliminated. Keep the needle valve open as far as possible to prevent sluggishness in the governor response, The needle valve setting varies from 1/16 to 2 turns open. Never close it tight, the governor cannot operate satisfactorily when this condition exists.
- 3. Check the governor stability by manually disturbing the governor speed setting. The compensation adjustment is satisfactory when the governor returns to speed with only a slight over- or undershoot. Once the needle valve adjustment is correct, it is not necessary to change the setting except for large, permanent changes in temperature which affect governor oil viscosity.

WARNING The engine, turbine, or other type of prime mover should be equipped with an overspeed shutdown device to protect against runaway or damage to the prime mover with possible personal injury, loss of life, or property damage.

The overspeed shutdown device must be totally independent of the prime mover control system. An overtemperature or overpressure shutdown device may also be needed for safety, as appropriate.

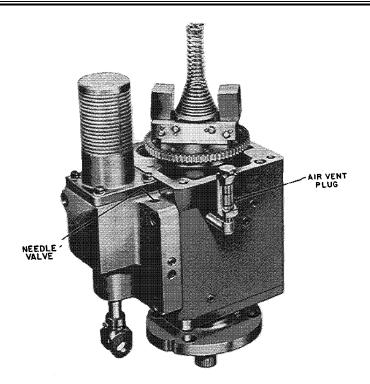


Figure 2-1. Adjustment Locations

Chapter 3. Maintenance

General

A governor should operate for several years before needing replacement if it is kept clean, and if the drive from the turbine or engine is smooth and free from torsional oscillations. Except for isolated cases, so rare they can be almost disregarded, governors do not fail or break down suddenly. Instead, they wear gradually, and give an external indication of their condition in the form of slight hunting, sluggish operation, etc. Further deterioration is at a slow enough rate so that an exchange governor may be ordered for installation at the next scheduled shutdown. Because there is so little chance of failure, we do not recommend keeping a complete stock of spare parts on hand. However, plants at which an unplanned shutdown would be very costly often keep a complete spare governor on hand.

It is advisable that the best mechanic available, preferably one experienced with small parts assembly, be permanently assigned to all governor repair work. Cleanliness of tools and work space is essential. A work bench, vise, arbor press, speed lathe, air supply, and containers for cleaning solvents should be provided, if possible. The usual small hand tools are required, and a few special Woodward tools may be desirable if sub-assemblies are to be disassembled.

Governor Oil

Contaminants and foreign matter in the governor oil are the greatest single source or cause of governor troubles. Use only new or filtered oil. Be sure that all containers used for the governor oil are clean. The time interval between governor oil changes depends on many factors: type of service, operating temperature, quality of oil, etc. Begin an oil maintenance program by inspecting the oil after three months' service. If inspection shows the oil to be satisfactory, gradually lengthen the time between inspections. Anytime the governor oil appears to be dirty or breaking down from contaminants or excessive temperatures, drain the governor while it is hot, flush with the lightest grade of the same oil, and refill with fresh oil. In any event, it is recommended that the oil be changed at least once every eighteen months.

Inspection and Test

Manual 36404 lists the most frequent indications of governor malfunctioning, and suggests possible causes and the corrective actions needed. A brief summary of these suggestions follows.

Governor faults are usually revealed in speed variations of the engine, but it does not necessarily follow that all such speed variations indicate governor faults. Therefore, when improper speed variations appear, make these checks:

- 1. Check the load to be sure that the speed changes observed are not the result of load changes.
- 2. Check the engine operation to be sure that all cylinders are firing properly, and that the injectors are in good operating condition.
- 3. See that the operating linkage between the governor and engine is free from binding or lost motion.

4. Check the setting of the compensating needle valve. It is impossible for the governor to function correctly if the needle valve is closed tight.

If neither load nor engine irregularities are found to be the cause of the speed variation, and adjustments of the needle valve are unable to correct the malfunctioning, the cause may be either in the governor or the drive to the governor.

If the speed variations are erratic but small in magnitude, the fault may lie in the drive to the governor. Excessive backlash or a tight meshing of the gears driving the governor may be the cause. No amount of adjustment or other work on the governor can correct this condition.

If the speed variations of the governor are large and erratic, and unaffected (except perhaps in magnitude) by changes of adjustment, or if the governor fails to control at all, it should be repaired or replaced.

As a last resort, to prove whether the engine or governor is at fault, another governor, known to be in good condition, should be installed on the engine. When this is done, the test governor must be of the exact same model as the one being removed.

Removal of Governor from Engine

In cases of major repairs or a complete governor change, the unit may be removed from the engine as follows:

- 1. Remove the oil drain plug. Drain oil from the governor and replace the drain plug. NOTE: The governor may be equipped with a drain cock for draining.
- 2. Disconnect linkage from the governor. If levers are removed from serrated shafts, mark their radial positions on the shafts so that they can be reinstalled in the same positions.
- 3. Disconnect, at the governor, other connections (electrical, pneumatic, hydraulic, etc.) to the governor.
- 4. Remove the four stud nuts holding the governor assembly to the governor mounting pad and lift the governor off the studs and away from the engine. Remove the gasket between the governor and governor mounting pad.

NOTICE

At all times use care in handling and setting the governor down. Be particularly careful to avoid striking the end of the drive shaft a sharp blow. Such treatment can damage the governor components.

Disassembly Procedures

An outline of the order of disassembly of the basic components follows. Hints for the disassembly of subassemblies are given where necessary. (The numbers in parentheses refer to the parts shown in Figure 4-1.)

1. Remove speeder spring (62).

- 2. Lift out the flyweight head-pilot valve bushing assembly. (items 63 through 101). After detaching the flyweight head (item 93), take out snap ring (81) to permit removal of items (82) and (85).
- 3. Remove snap ring (60) and items (55) through (59).
- 4. Remove four screws (8) and lockwashers (9) to detach the power cylinder assembly (items 5 through 22).
 - a. To remove spring guard (20), position the power cylinder assembly in an arbor press as shown in Figure 3-1. Use the arbor press to hold the spring guard down against the force of power spring (17) while removing screws (22) and washers (21).

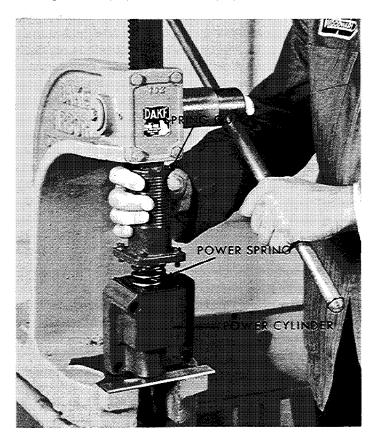


Figure 3-1

PG Governor Basic Elements

- b. Use a rod end puller tool (Woodward tool 012281) to remove fuel rod end (5). See Figure 3-2 for use of the puller.
- c. Grip power cylinder (10) in a vise to remove oil seals as shown in Figure 3-3.

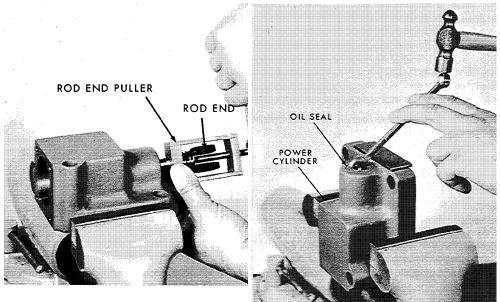


Figure 3-2

Figure 3-3

5. Remove screws (53) and retainer (52). Take out drive shaft (51) and items (46) through (50). Remove drive shaft oil seal (47) as shown in Figure 3-4.



Figure 3-4

- 6. Remove screws (45) and washers(44). Lift off base (43) taking care that idler gear (40) and drive gear (38) do not fall out. Remove the idler gear and drive gear.
- 7. Set the power case (28), bottom end (end with idler stud 39) down, in an arbor press or small drill press. With a rod against spring seat (24), compress accumulator springs (25) and (26) to permit removal of upper snap ring (23). See Figure 3-5. Remove spring seat (24) and springs (25) and (26).

Invert the power case and remove snap rings (23) and accumulator pistons (37).

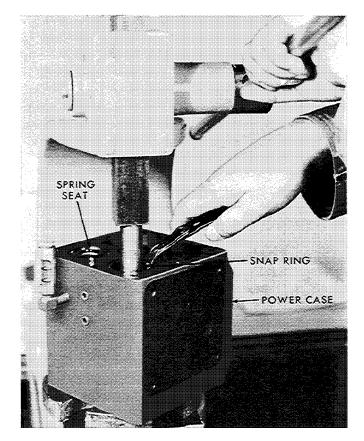


Figure 3-5

- 8. If necessary to remove check valve assemblies (29 and 30), proceed in this manner:
 - a. To remove inner check valves (29), pry the retainer plate from the check valve assembly and remove springs and check balls.
 - b. To remove outer check valves (30), press the check balls through and out of the valve case.

- c. Then tap all four check valve cases with a 1/4"-28 tap. Using a 1/4"-28 bolt with a small plate as a jack, pull the four valve cases. See Figure 3-6.
- d. Remove two balls from the power case.

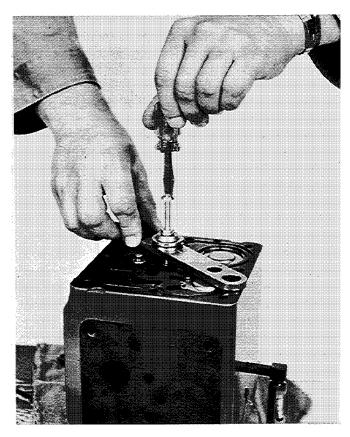


Figure 3-6

Inspection after Disassembly

After disassembling the governor, wash all parts in clean fuel oil and carefully inspect for wear. Generally, most of the repair work consists of cleaning and polishing the governor parts. All pistons, plungers, valves, and rods should move freely without binding or catching. Do not lap in parts if possible to free up by other means.

Inspect the check valves (if used) in the bottom of the governor power case. They must be clean, operating freely, and seating properly.

The flat, joint surfaces at top and bottom of both governor case and base, must be free of burrs and high spots.

Carefully avoid scratching or scoring these faces, particularly the top of the base and the bottom of the case. Avoid damage to the neoprene seal ring and the groove in the bottom face of the governor power case. The pilot valve plunger should move freely in the pilot valve bushing assembly. If the plunger surfaces are scratched, they may be dressed with a hard Arkansas stone.

NOTICE	The edges of the control land and compensating piston MUST be left sharp.
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The power piston assembly should move freely in the power cylinder, and the buffer piston should move freely in the buffer cylinder.

Ballhead flyweights must work freely on the needle bearings and pivot pins.

If the toes of the flyweights have flat spots, it will be necessary to disassemble the ballhead and round off the worn spots with emery cloth. If wear is excessive, install new flyweights.

Ball races and bearing balls of the flyweight thrust bearing should be in perfect condition.

Assembly Procedures

While assembling the parts into the governor, take care to ensure that no lint or other foreign matter is present on the parts. The governor may be assembled dry, or if preferred a small amount of clean lubricating oil can be applied to the parts as they are placed into the governor. Replace all pipe plugs removed from the governor, using a good joint compound on the threads of the plug—NOT IN THE HOLE. When the governor is assembled, apply a liberal amount of clean lubricating oil over all the moving parts, to ensure initial lubrication.

The order of assembly is, in general, the reverse of the disassembly process. The following hints should aid in assembly of the basic elements. (The numbers shown in parentheses refer to the parts shown in Figure 4-1.)

- Use Woodward tool 360689 to press check valve assembly (29) into power case (28). Use Woodward tool 360690) to press in check valve assembly (30).
- After installing lower snap rings (23), accumulator pistons (37), and springs (25) and (26), use a rod and an arbor press to compress spring seats (24). Install upper snap rings (23).
- 3. With the power case components installed and the power case resting upside down (with idler gear stud 39 up), install idler gear (40), drive gear (38), and oil seal ring (41). Be sure the gears turn freely.

Attach base (43) to the power case with screws (45). Do nor tighten the screws. Insert drive shaft (51) with bearing (50) and snap ring (49) already on the drive shaft into the base and turn until the splined end slips into the splines in the pump drive gear. Continue turning to check alignment and free rotation of the drive gear and idler gear while tightening screws (45).

Withdraw the drive shaft from the base.

4. Press oil seal (47) into oil seal retainer (48) flush with the end opposite the flange end of retainer.

PG Governor Basic Elements

- Install gasket (46) and the assembly consisting of the oil seal and oil seal retainer. Install the drive shaft assembly. Attach retainer (52) with screws (53). Tighten screws evenly. Do not attempt to bring the retainer into contact with the base. Install lock wire (54) on screws (53).
- 6. Press oil seals (6) and (7) into power cylinder (10). Seal (7) should be inserted with the part number facing out. Seal (6) should be inserted with the part number facing in and pressed in to 0.005" (0.13 mm) below flush.
- 7. Place oil seal inserter (Woodward tool 360066) on over rod of power piston assembly (16) to avoid damaging oil seals (6) and (7) when inserting the piston assembly in the power cylinder.
- 8. Position the power cylinder in an arbor press, align the taper pin holes, and press rod end (5) onto the end of power piston (16). See Figure 3-7. Place a small steel block in the rod end slot so that pressure is applied against the bottom of the slot.

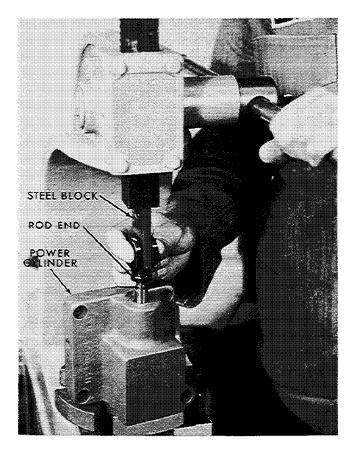


Figure 3-7

- 9. When assembling the power cylinder assembly onto the power case, align the holes in gasket (13) with the holes in power case (28) instead of with those in the power cylinder.
- 10. When assembling the flyweight head pilot valve bushing assembly, align the missing tooth in the pilot valve bushing with the corresponding missing tooth in its mating part (items 76 and 87).

- 11. The three-piece thrust bearing (66) fits onto the stem of the pilot valve plunger (85) with the bearing race with the larger hole on the bottom (against the flyweight toes).
- 12. "Center" the pilot valve plunger in this manner: wish slight pressure on the speeder spring seat (65), adjust the pilot valve plunger nut (63) until, as the flyweights (70) are moved from their extreme inward to their extreme outward position, there is the same amount of control land showing in the control port at each extreme.

The control ports are the bottom row of holes in the pilot valve bushing assembly. See Figure 3-8. (Note: Shutdown rod (72) and retaining ring (73), if used, must be inserted through nut (63) before the centering adjustment is made.) Be sure to install cotter pin (64).

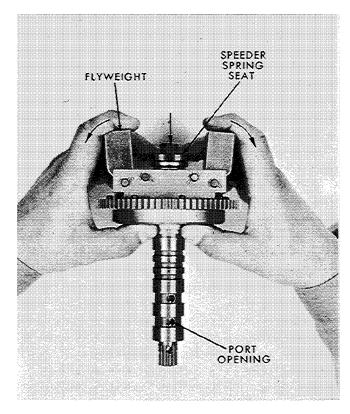


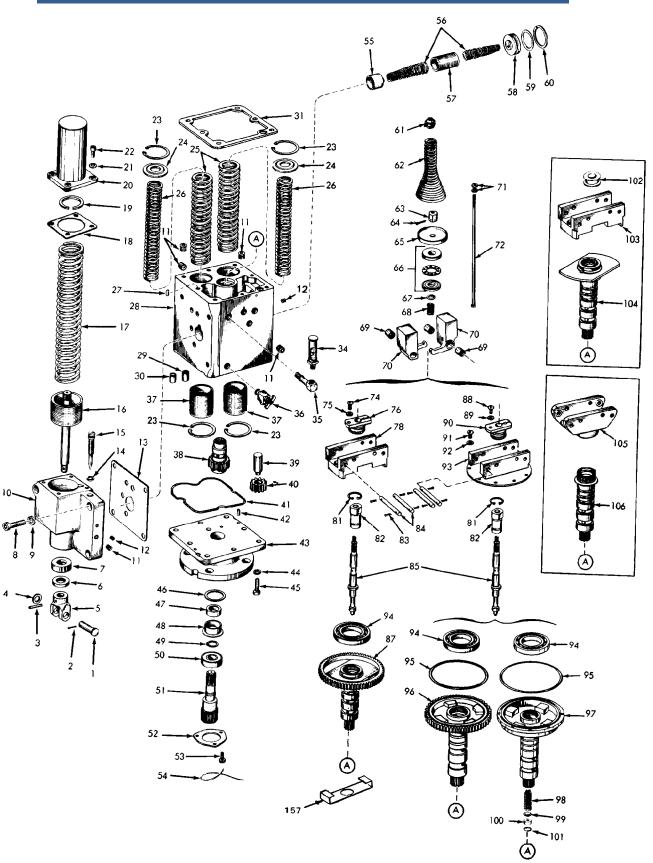
Figure 3-8

Chapter 4. Replacement Parts

When ordering replacement parts, include the following information:

- Governor serial number and part number shown on the nameplate
- Manual number (this is manual 36600)
- Part reference number in parts list and description of part or part name.

Ref. No.	Part NameQuantity	Ref. No.	Part Name Quantity
36600-1	Rod End Pin1	36600-57	Buffer Piston1
36600-2	Cotter Pin1	36600-58	Plug1
36600-3	Taper Pin1	36600-59	O-ring1
36600-4	Rod End Pin Washer1	36600-60	Snap Ring 1
36600-5	Rod End1	36600-61	Speeder Spring Check Plug1
36600-6	Oil Seal1	36600-62	Speeder Spring1
36600-7	Oil Seal1	36600-63	Pilot Valve Plunger Nut1
36600-8	Socket Head Screw (3/8"-16 x 1-1/4)4	36600-64	Cotter Pin1
36600-9	Split Lockwasher (3/8")4	36600-65	Speeder Spring Seat1
36600-10	Power Cylinder1	36600-66	Thrust Bearing1
36600-11	Pipe Plug (1/8")as req.	36600-67	Adjusting Spring Washer1
36600-12	Pipe Plug (1/16") as req.	36600-68	Adjusting Spring1
36600-13	Gasket1	36600-69	Flyweight Bearing
36600-14	O-ring1	36600-70	Flyweight2
36600-15	Needle Valve1	36600-71	Shutdown Nut
36600-16	Power Piston and Rod Assembly1	36600-72	Shutdown Rod1
36600-17	Power Cylinder Spring1	36600-73	Not Used
36600-18	Gasket1		upled–Undamped Flyweight Head Assy
36600-19	Piston Stop Ring1	36600-74	Round Head Screw (#6-32 x 5/16")1
36600-20	Spring Guard1	36000-75	Split Lockwasher (#6) 1
36600-20	Shakeproof Washer (1/4")4	36600-76	Spring Coupling Assembly1
36600-22	Socket Head Screw (1/4"-28 x 3/4)4	36600-77	Not Used
36600-22	Snap Ring	36600-77	Flyweight Head1
36600-23	Shap King4 Spring Seat	36600-78	Not Used
36600-24			Not Used
36600-25	Accumulator Spring (Large)2	36600-80	
	Accumulator Spring (Small)2	36600-81	Snap Ring1
36600-27	Case-to-Column Dowel Pin2	36600-82	Compensating Bushing1
36600-28	Power Case1	36000-83	Cotter Pin
36600-29	Check Valve Assembly (Spring Loaded)2	36000-84	Flyweight Pin–Limit Pin
36600-30	Check Valve Assembly (Plain)2	36000-85	Pilot Valve Plunger
36600-31	Gasket1	36600-87	Flyweight Hd Gear-Pilot Vlv Bshng Assy 1
36600-32	Not Used	36600-94	Centering Bearing
36600-33	Not Used		upled— Oil Damped Flyweight Head Assy
36600-34	Oil Level Gauge1	36600-88	Round Head Screw (#8-32 x 5/16)1
36600-35	Elbow1	36600-89	Split Lockwasher (#8)1
36600-36	Drain Cock2	36600-90	Spring Coupling Assembly1
36600-37	Accumulator Piston1	36600-91	Fillister I-lead Screw (#5-40 x 9/32)8
36600-38	Drive Gear1	36600-92	Lockwasher (#5)8
36600-39	Idler Gear Stud1	36600-93	Flyweight Head1
36600-40	Idler Gear1	36600-94	Centering Bearing1
36600-41	Oil Seal Ring1	36600-95	Oil Seal Ring1
36600-42	Case-to-Base Dowel Pin2	36600-96	Flyweight Hd Gear-Pilot Vlv Bshng Assy 1
36600-43	Base1	36600-97	Flyweight Hd Cup-Pilot Vlv Bshng Assy1
36600-44	Split Lockwasher (21/64")8		pecial Low Speed Governor Operation
36600-45	Hex Head Screw (5/16"-18 x 1")8	36600-98	Spring1
36600-46	Gasket1	36600-99	Spring Seat1
36600-47	Oil Seal1	36600-100	Plug1
36600-48	Oil Seal Retainer1	36600-101	Snap Ring1
36600-49	Snap Ring1	Rubber Da	mped Flyweight Head Assembly
36600-50	Bearing1	36600-102	Rubber Coupling Assembly1
36600-51	Drive Shaft1	36600-103	Flyweight Head1
36600-52	Bearing Retainer1	36600-104	
36600-53	Hex Head Screw (1/4"-28 x 5/8)		eight Head Assembly
36600-54	Lockwire as req.	36600-105	
36600-55	Spring Seat	36600-106	Pilot Valve Bushing Assembly1
36600-56	Buffer Spring		
	-1		





Power Cylinder Assembly with Rotary Terminal Shaft 36600-107 Power Piston and Rod Assembly......1 36600-108 Drive Screw2 Terminal Shaft Scale1 36600-109 Rack Dial Segment1 36600-110 Taper Pin1 36600-111 Socket Head Screw (#10-32 x 5/8")......1 36500-112 36600-113 Split Lockwasher (#10)1 36600-114 Oil Seal (Small).....1 36600-115 Needle Bearing (Small).....1 Socket Head Screw (#8-32 x 3/4")......2 36600-116 36600-117 Split Lockwasher (#8)2 36600-118 Compensating Needle Valve1 36600-119 O-ring......1 36600-120 Split Lockwasher (#3/8") 4 Socket Head Screw (3/8"-16 x 1")......1 36600-121 Rack Scale Pointer1 36600-122 Snap Ring (Large)2 36600-123 36600-124 36600-125 36600-126 Stop Nut (7/16"-20) 1 36600-127 Power Lever......1 36600-128 Clamp Screw2 36600-129 36600.130 Gasket1 36600-131 Cover 1 36600-132 Split Lockwasher......8 36600-133 Pipe Plug (1/8" NPT).....1 36600-134 36600-135 Gasket1 Power Cylinder1 36600-136 Needle Bearing (Large)1 36600-137 36600-138 Oil Seal (Large).....1 36600-139 36600-140 Piston Rod Pin1 Power Lever Pin1 36600-141 36600-142 Snap Ring (Small).....2

Alternate Base Assembly

36600-143	Base1
36600 144	Gasket1
36600-145	Oil Seat1
36600-146	Oil Seal Retainer1
36600-147	Serrated Drive Shaft (Long)1
36600-148	Bearing1
36600-149	Snap Ring1
36600-150	Bearing Retainer1
36600-151	Hex Head Screw (1/4"-28 x 5/8")
36600-152	Keyed Drive Shaft1
36600-153	Straight Key1
36600-154	Spacer Sleeve1
36600-155	Castle Nut (5/8"-18) 1
36600-156	Lockwireas req.
36600-157	Retainer Clip

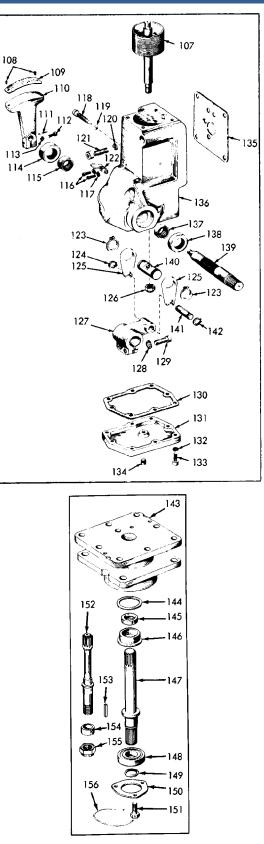


Figure 4-2. Exploded View, Alternate Power Cylinder and Base Assemblies

Chapter 5. Product Support and Service Options

Product Support Options

If you are experiencing problems with the installation, or unsatisfactory performance of a Woodward product, the following options are available:

- 1. Consult the troubleshooting guide in the manual.
- 2. Contact the **OE Manufacturer or Packager** of your system.
- 3. Contact the **Woodward Business Partner** serving your area.
- 4. Contact Woodward technical assistance via email (EngineHelpDesk@Woodward.com) with detailed information on the product, application, and symptoms. Your email will be forwarded to an appropriate expert on the product and application to respond by telephone or return email.
- 5. If the issue cannot be resolved, you can select a further course of action to pursue based on the available services listed in this chapter.

OEM or Packager Support: Many Woodward controls and control devices are installed into the equipment system and programmed by an Original Equipment Manufacturer (OEM) or Equipment Packager at their factory. In some cases, the programming is password-protected by the OEM or packager, and they are the best source for product service and support. Warranty service for Woodward products shipped with an equipment system should also be handled through the OEM or Packager. Please review your equipment system documentation for details.

Woodward Business Partner Support: Woodward works with and supports a global network of independent business partners whose mission is to serve the users of Woodward controls, as described here:

- A **Full-Service Distributor** has the primary responsibility for sales, service, system integration solutions, technical desk support, and aftermarket marketing of standard Woodward products within a specific geographic area and market segment.
- An **Authorized Independent Service Facility (AISF)** provides authorized service that includes repairs, repair parts, and warranty service on Woodward's behalf. Service (not new unit sales) is an AISF's primary mission.
- A **Recognized Engine Retrofitter (RER)** is an independent company that does retrofits and upgrades on reciprocating gas engines and dual-fuel conversions, and can provide the full line of Woodward systems and components for the retrofits and overhauls, emission compliance upgrades, long term service contracts, emergency repairs, etc.

A current list of Woodward Business Partners is available at www.woodward.com/directory.

Product Service Options

Depending on the type of product, the following options for servicing Woodward products may be available through your local Full-Service Distributor or the OEM or Packager of the equipment system.

- Replacement/Exchange (24-hour service)
- Flat Rate Repair
- Flat Rate Remanufacture

Replacement/Exchange: Replacement/Exchange is a premium program designed for the user who is in need of immediate service. It allows you to request and receive a like-new replacement unit in minimum time (usually within 24 hours of the request), providing a suitable unit is available at the time of the request, thereby minimizing costly downtime.

This option allows you to call your Full-Service Distributor in the event of an unexpected outage, or in advance of a scheduled outage, to request a replacement control unit. If the unit is available at the time of the call, it can usually be shipped out within 24 hours. You replace your field control unit with the like-new replacement and return the field unit to the Full-Service Distributor.

Flat Rate Repair: Flat Rate Repair is available for many of the standard mechanical products and some of the electronic products in the field. This program offers you repair service for your products with the advantage of knowing in advance what the cost will be.

Flat Rate Remanufacture: Flat Rate Remanufacture is very similar to the Flat Rate Repair option, with the exception that the unit will be returned to you in "like-new" condition. This option is applicable to mechanical products only.

Returning Equipment for Repair

If a control (or any part of an electronic control) is to be returned for repair, please contact your Full-Service Distributor in advance to obtain Return Authorization and shipping instructions.

When shipping the item(s), attach a tag with the following information:

- return number;
- name and location where the control is installed;
- name and phone number of contact person;
- complete Woodward part number(s) and serial number(s);
- description of the problem;
- instructions describing the desired type of repair.

Packing a Control

Use the following materials when returning a complete control:

- protective caps on any connectors;
- antistatic protective bags on all electronic modules;
- packing materials that will not damage the surface of the unit;
- at least 100 mm (4 inches) of tightly packed, industry-approved packing material;
- a packing carton with double walls;
- a strong tape around the outside of the carton for increased strength.

NOTICE To prevent damage to electronic components caused by improper handling, read and observe the precautions in Woodward manual 82715, *Guide for Handling and Protection of Electronic Controls, Printed Circuit Boards, and Modules.*

Replacement Parts

When ordering replacement parts for controls, include the following information:

- the part number(s) (XXXX-XXXX) that is on the enclosure nameplate;
- the unit serial number, which is also on the nameplate.

Engineering Services

Woodward's Full-Service Distributors offer various Engineering Services for our products. For these services, you can contact the Distributor by telephone or by email.

- Technical Support
- Product Training
- Field Service

Technical Support is available from your equipment system supplier, your local Full-Service Distributor, or from many of Woodward's worldwide locations, depending upon the product and application. This service can assist you with technical questions or problem solving during the normal business hours of the Woodward location you contact.

Product Training is available as standard classes at many Distributor locations. Customized classes are also available, which can be tailored to your needs and held at one of our Distributor locations or at your site. This training, conducted by experienced personnel, will assure that you will be able to maintain system reliability and availability.

Field Service engineering on-site support is available, depending on the product and location, from one of our Full-Service Distributors. The field engineers are experienced both on Woodward products as well as on much of the non-Woodward equipment with which our products interface.

For information on these services, please contact one of the Full-Service Distributors listed at <u>www.woodward.com/directory</u>.

Contacting Woodward's Support Organization

For the name of your nearest Woodward Full-Service Distributor or service facility, please consult our worldwide directory published at <u>www.woodward.com/directory</u>.

You can also contact the Woodward Customer Service Department at one of the following Woodward facilities to obtain the address and phone number of the nearest facility at which you can obtain information and service.

Products Used In Electrical Power Systems	Products Used In Engine Systems	Products Used In Industrial Turbomachinery Systems
FacilityPhone Number	<u>Facility</u> <u>Phone Number</u>	FacilityPhone Number
Brazil+55 (19) 3708 4800	Brazil+55 (19) 3708 4800	Brazil+55 (19) 3708 4800
China +86 (512) 6762 6727	China +86 (512) 6762 6727	China +86 (512) 6762 6727
Germany:	Germany +49 (711) 78954-510	India+91 (129) 4097100
Kempen+49 (0) 21 52 14 51	India+91 (129) 4097100	Japan +81 (43) 213-2191
Stuttgart +49 (711) 78954-510	Japan +81 (43) 213-2191	Korea +82 (51) 636-7080
India+91 (129) 4097100	Korea +82 (51) 636-7080	The Netherlands- +31 (23) 5661111
Japan +81 (43) 213-2191	The Netherlands- +31 (23) 5661111	Poland+48 12 295 13 00
Korea +82 (51) 636-7080	United States +1 (970) 482-5811	United States +1 (970) 482-5811
Poland+48 12 295 13 00		
United States +1 (970) 482-5811		

For the most current product support and contact information, please visit our website directory at <u>www.woodward.com/directory</u>.

Technical Assistance

If you need to contact technical assistance, you will need to provide the following information. Please write it down here before contacting the Engine OEM, the Packager, a Woodward Business Partner, or the Woodward factory:

General	
Your Name	
Site Location	
Phone Number	
Fax Number	
Prime Mover Information	
Manufacturer	
Engine Model Number	
Number of Cylinders	
Type of Fuel (gas, gaseous, diesel, dual-fuel, etc.)	
Power Output Rating	
Application (power generation, marine, etc.)	
Control/Governor Information	
Control/Governor #1	
Woodward Part Number & Rev. Letter	
Control Description or Governor Type	
Serial Number	
Control/Governor #2	
Woodward Part Number & Rev. Letter	
Control Description or Governor Type	
Serial Number	
Control/Governor #3	
Woodward Part Number & Rev. Letter	
Control Description or Governor Type	
Serial Number	
Symptoms	
Description	

If you have an electronic or programmable control, please have the adjustment setting positions or the menu settings written down and with you at the time of the call.

Revision History

Changes in Revision H—

• Revision advanced to coordinate with new installation sheet.

We appreciate your comments about the content of our publications.

Send comments to: icinfo@woodward.com

Please reference publication 36600H.



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